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Diesel Fuel Lubricity Requirements for Light Duty Fuel Injection Equipment

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Diesel Fuel Lubricity Requirements for Light Duty Fuel Injection Equipment

CARB Fuels Workshop

**Sacramento, CA
Feb. 20, 2003**

**Klaus Meyer and Thomas C. Livingston
Robert Bosch GmbH**



This presentation covers the interests of

- Robert Bosch GmbH**
- Delphi Diesel Systems**
- Denso Corporation**
- SiemensVDO Automotive AG**
- Stanadyne Automotive Corporation**



**Our Mission is to increase the number of
Diesel vehicles in the USA
especially Passenger Cars + SUVs + Light Duty (LD)**

- Build a Cleaner Environment
- Conserve Energy Resources
- Reduce Fuel Consumption / CAFE
 - Lower CO₂ Emission

- For Diesel Fuel Injection Equipment (DFIE)
Lubricity
is the most valuable and crucial property



Scope of Presentation

Introduction

Experience in Europe

Comparing USA and Europe

- Vehicles and DFIE
- Survey Data

Requirements

- HFRR method
- Sensitivity of DFIE to HFRR
- Rating Table for Assessed Pump Wear
- Endurance Performance

Data for Diesel Fuels with HFRR between 400 - 650 μm

- Pump Performance: Rotary pumps, Common Rail Systems

Engine Results

Conclusions



A Brief Review

- Sweden introduced sulphur-free fuels in 1990, California followed in 1993 with low-sulphur fuels
 - Failures of fuel-lubricated injection pumps (for passenger and light duty vehicles)
- Lubricity identified as cause
 - Hydroprocessing for desulphurization reduces lubricity enhancing fuel components
- All DFIE manufacturers **afflicted**
- Process to define wear test method and lubricity limit for fuel spec:
HFRR (ISO 12156-1, -2, ASTM D-6079)
® **EN 590 et al. Lubricity Limit =460 µm**
SLBOCLE (ASTM D-6078/99)



Current Situation in EU

- In Europe **40 %** of new cars are **Diesel** vehicles:
 - Passenger and Light Duty vehicles (e.g. SUV)
- EN 590 lubricity spec. (**HFRR 460 µm max.**) successfully prevents field problems
- Diesel vehicles improve fuel consumption **by 30 %** compared to SI engines
- Diesel vehicles have low fuel consumption (**up to 78 mpg**)
- Diesel vehicles produce **lower CO₂** emissions
- Diesel vehicles provide low service costs and high service intervals
- Drivers enjoy driving diesel vehicles due to superior torque characteristics
- Majority of High Pressure DFIE is **fuel-lubricated**



Main Differences in Diesel Vehicles

| | Today ↓ | | Future ↓ | |
|--------------------------------|--|--|---|--|
| | U.S. / California | EU | U.S. / California | EU |
| Vehicles | <ul style="list-style-type: none">• Heavy Duty• Light Duty | <ul style="list-style-type: none">• Passenger• Light Duty• Heavy Duty | <ul style="list-style-type: none">• Light Duty• Heavy Duty• Passenger | <ul style="list-style-type: none">• Passenger• Light Duty• Heavy Duty |
| DFIE | <ul style="list-style-type: none">• Inline pumps• UIS/UPS• Common Rail• Rotary pumps | <ul style="list-style-type: none">• Common Rail• UIS/UPS• Rotary pumps | <ul style="list-style-type: none">• Inline pumps• UIS/UPS• Common Rail• Rotary pumps | <ul style="list-style-type: none">• Common Rail• UIS/UPS• Rotary pumps |
| Lubricity requirement | (+) | ++ | ++ | ++ |
| Lubricity specification | U.S.A.: none CA: SLBOCLE guideline | HFRR 460 µm max. | HFRR 460 µm max. | HFRR 460 µm max. |

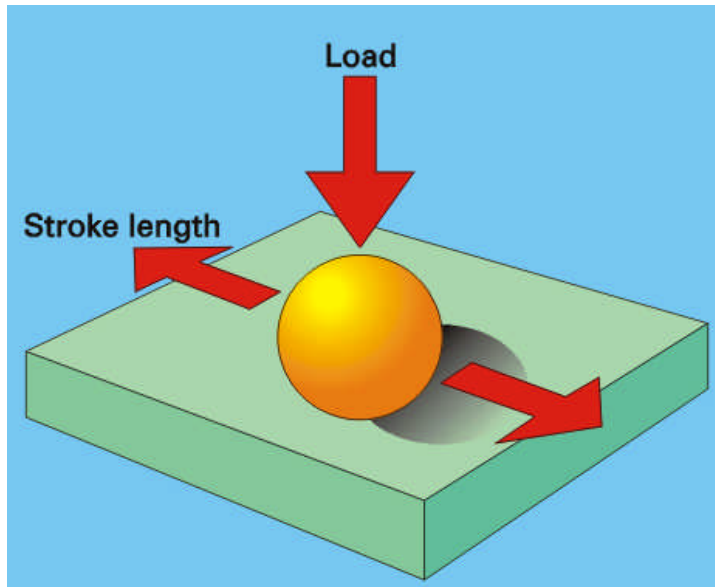


Samples from Summer 2002

| Property | Unit | U.S.A. | Europe (EN 590) | Assessment of U.S.A. Quality |
|---------------------------------------|-------------------|-------------|--------------------|---|
| Density | kg/m ³ | 813 ... 863 | 820 ... 845 | wide range |
| Viscosity | c.St. (40 °C) | 2.1 ... 3.2 | 2.0 ... 4.5 | o.k. |
| Dist. 95% vol rec. | °C | 324 ... 344 | < 360 | o.k. |
| Total Aromatic Cont. | % | 16 ... 46 | n.a. | many high numbers |
| Cetane No. | | 44 ... 57 | > 51 | many low numbers |
| Sulphur | mg/kg | 23 ... 416 | < 350 | not o.k. for aftertreatment |
| Water | mg/kg | 42 ... 96 | < 200 | o.k. |
| Total Contamination (particulates) | mg/kg | 0.8 ... 3.1 | < 24 | some high numbers (EN590 limit too high) |
| Lubricity | µm (HFRR 60C) | 351 ... 648 | < 460 | 80% of samples > 460 µm |
| Alcohol | % vol. | < 0.1 | n.a. | o.k. |

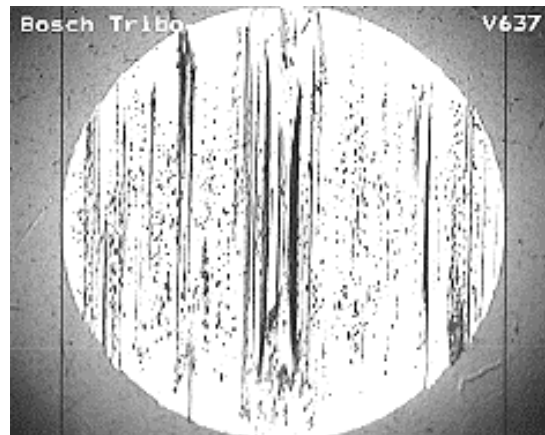


ISO 12156-1 Method



Test conditions:

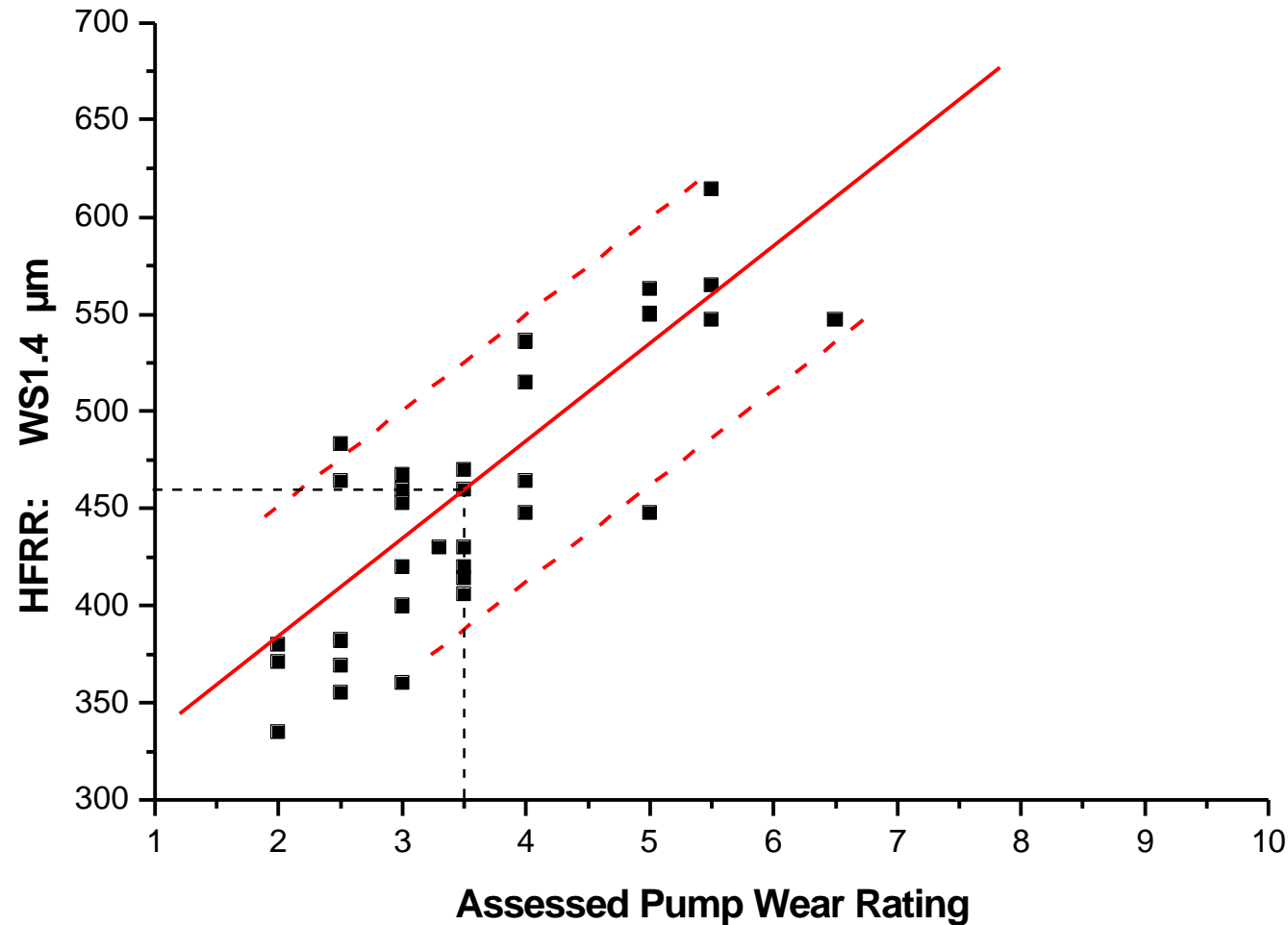
| | |
|-------------------|-------------------------------------|
| Applied load | $200 \text{ g} \pm 0.01 \text{ g}$ |
| Stroke length | $1 \pm 0.02 \text{ mm}$ |
| Frequency | $50 \pm 1 \text{ Hz}$ |
| Test duration | $75 \pm 0.1 \text{ min}$ |
| Fluid temperature | $60 \pm 2 \text{ }^{\circ}\text{C}$ |
| Fluid volume | $2 \pm 0.20 \text{ ml}$ |
| Bath surface | $6 \pm 1 \text{ cm}^2$ |



WS1.4 μm



Sensitivity of DFIE to HFRR



→ Linear regression: Pump wear 3.5 ==> WS1.4 = 454 μm



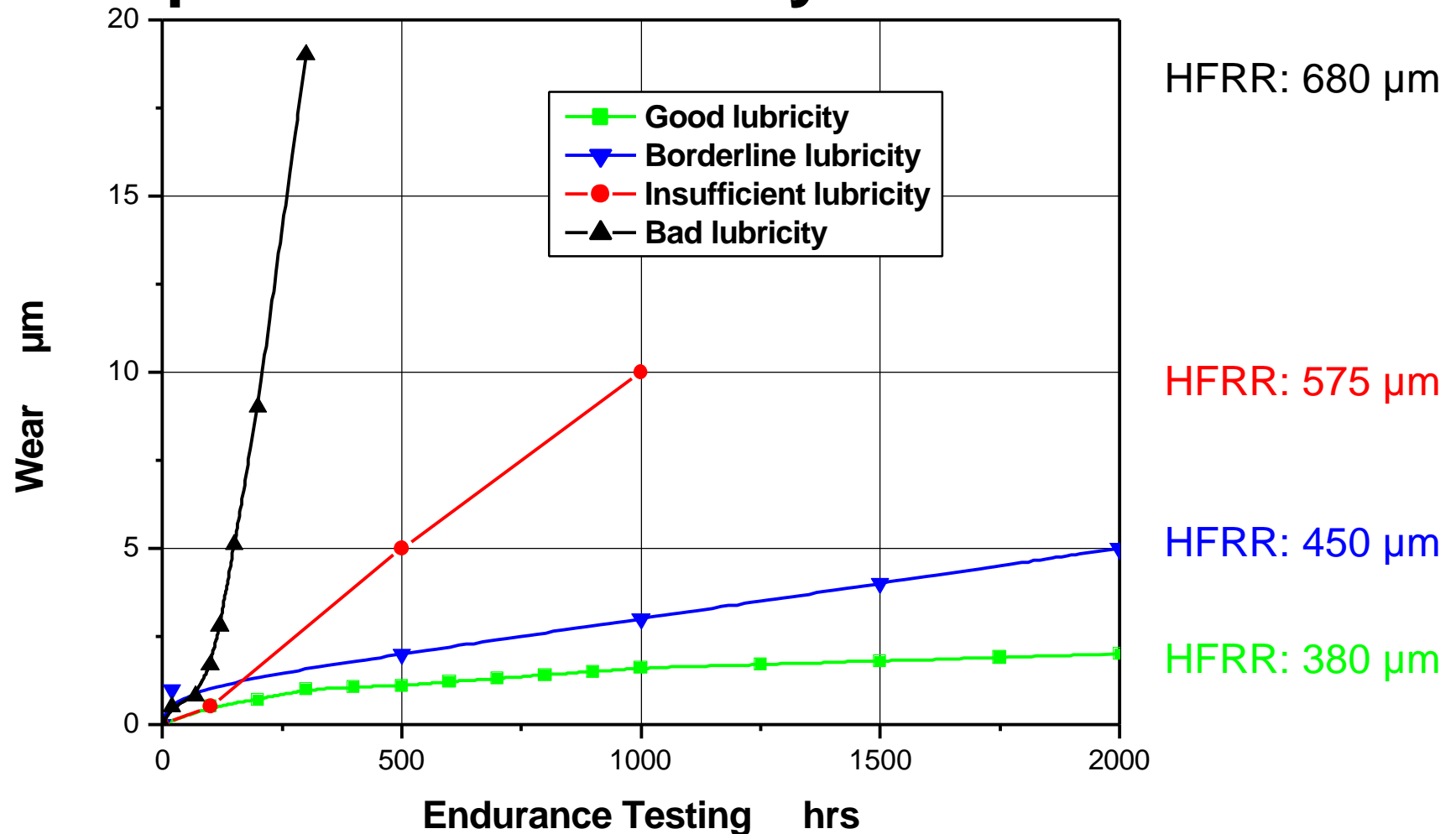
Table to Assess Pump Wear

| Component | Wear rating: 1 – 3.5 Durability + performance = 100 % | | Wear rating: 4 – 6 Durability reduced to 20 % | | Wear rating: 7 – 10 Durability reduced to 1 % Immediate failure | |
|-----------------------------------|--|-----------|--|-------------|---|------------------|
| | Type of wear | Wear rate | Type of wear | Wear rate | Type of wear | Wear rate |
| Cam plate | | | | | | |
| runway | rolling and abrasive | < 1 µm | seizure and fatigue | 1 – 30 µm | fatigue | not determinable |
| cam plate centre | fretting | 1 - 3 µm | fretting | 3 - 10 µm | fretting | > 10 µm |
| cam plate claws | fretting | < 10 µm | rolling and fretting | 10 - 20 µm | seizure | not determinable |
| Roller | rolling | < 1 µm | seizure and fatigue | 1 - 5 µm | seizure and fatigue | not determinable |
| Roller bolt | | | | | | |
| - point of contact to roller | rolling | < 1 µm | fretting and seizure | 1 - 10 µm | seizure | >10 µm |
| - point of contact to roller ring | fretting | < 10 µm | fretting | 10 - 15 µm | seizure | >15 µm |
| Fuel pump | | | | | | |
| - blades | fretting | < 10 µm | fretting | 10 - 200 µm | fretting and seizure | not determinable |
| - raceway | fretting | 1 – 2 µm | fretting | 2 - 100 µm | fretting and seizure | not determinable |

→ Pump wear must not exceed “green” zone to meet customer expectation



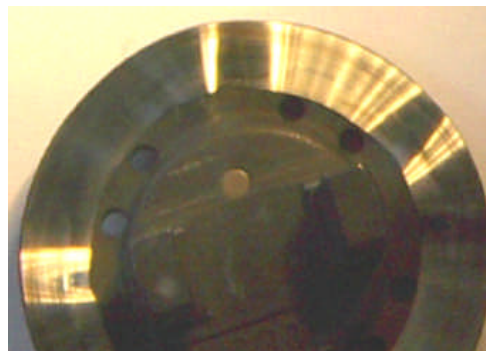
Pump Wear vs. Lubricity over Lifetime



→ New DFIE designed to operate with “blue --” fuel



VE - Rotary Pump with HFRR 450 μm Fuel



Wear rating = 3.5

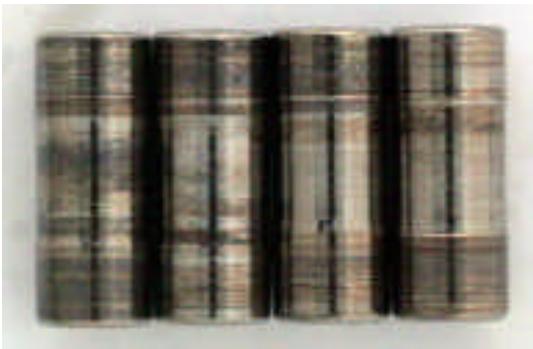
Bolts: **slight scuffing** Supply pump vanes: increased abrasive wear

→ **Fuel represents borderline EU quality**

→ **Fuel adequate for purpose**



VE - Rotary Pump with **HFRR 650 μm Fuel**



Wear rating = 8

Cam plate: 30 μm

Rollers: Seizure

Bolt: 15 μm

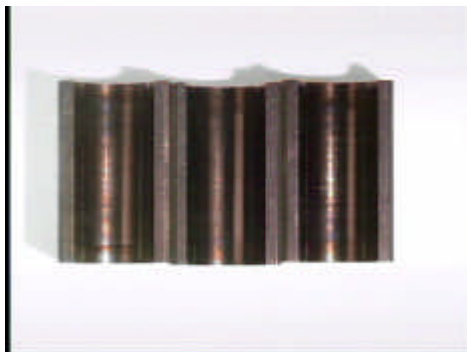
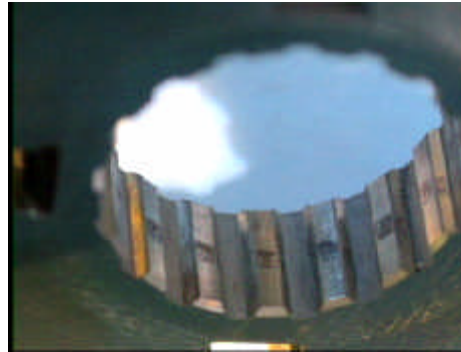
Piston: Broken

→ **Fuel represents worst case U.S. lubricity**

→ **Fuel unfit for purpose**



VP44 - Rotary Pump with HFRR 400 μm Fuel



Wear rating = 3.0

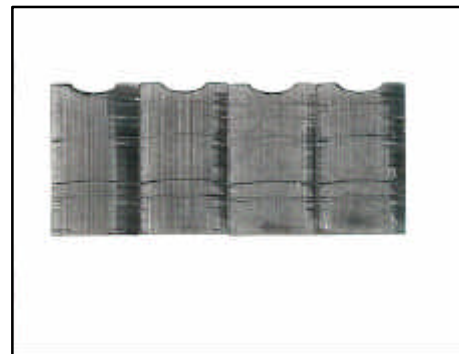
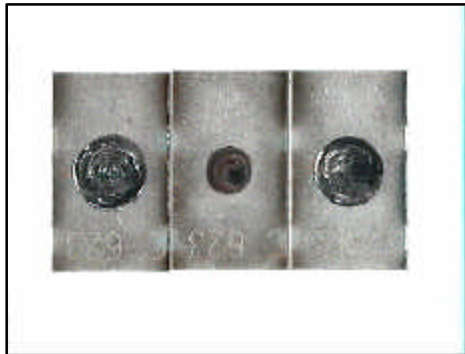
Supply pump, roller shoes, feed pump tooth system, and timing piston: minor polishing

→ **Fuel represents typical EU quality**

→ **Fuel fit for purpose**



VP44 - Rotary Pump with **HFRR 650 μm Fuel**



Wear rating = 7.0

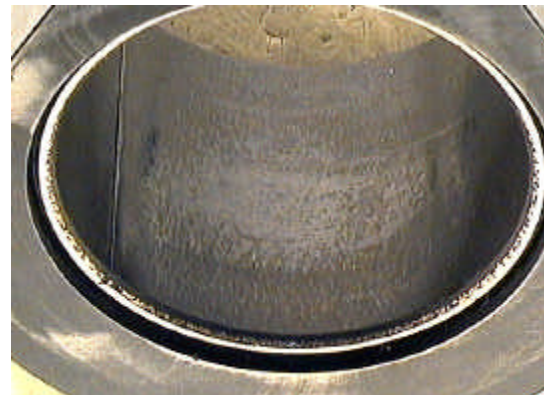
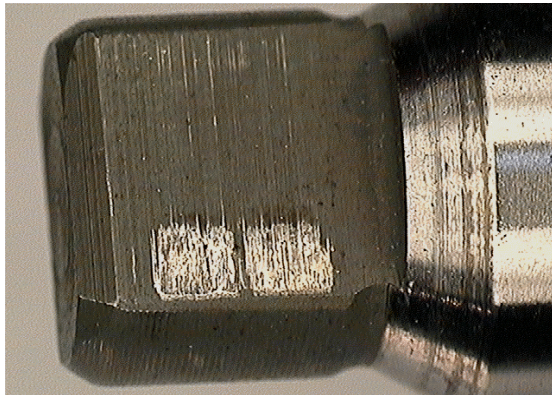
Supply pump, feed pump tooth system, high pressure piston and vanes: severe wear

→ **Fuel represents worst case U.S. lubricity**

→ **Fuel unfit for purpose**



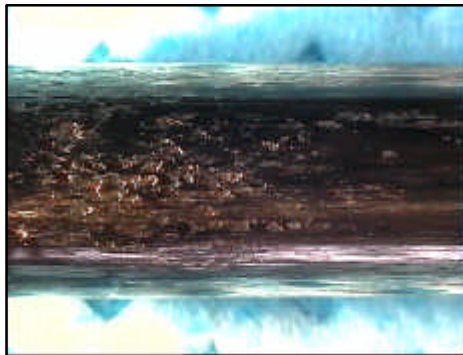
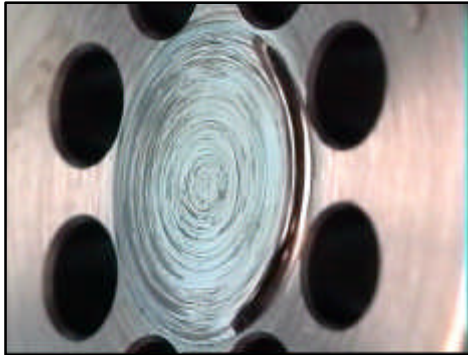
Common Rail System with HFRR 460 μm Fuel



- Fuel represents borderline EU quality
- Fuel adequate for purpose



Common Rail System with **HFRR 650 μm Fuel**



Wear rating = 9.0

Piston: Seizure Piston bottom center: 15 μm ; Bearing shell: Seizure; Polygon: $\geq 1000 \mu\text{m}$

→ **Fuel represents worst case U.S. lubricity**

→ **Fuel unfit for purpose**

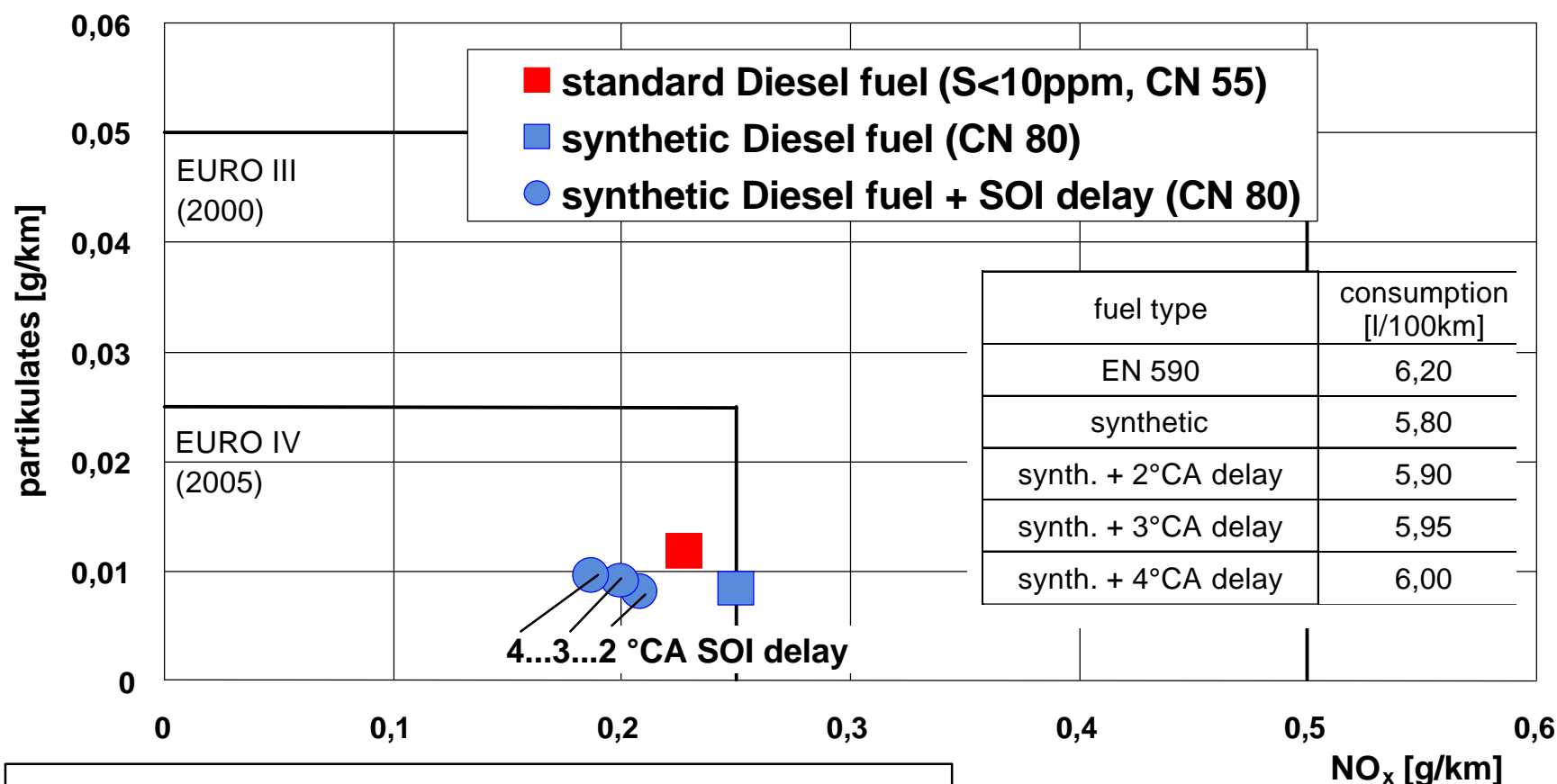


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| Total Contamination (particulates) | mg/kg | 0.8 ... 3.1 | < 24 | some high numbers (EN590 limit too high) |
| Lubricity | µm (HFRR 60C) | 351 ... 648 | < 460 | 80% of samples > 460 µm |
| Alcohol | % vol. | < 0.1 | n.a. | o.k. |



NO_x and PM Reduction with CN 55 and 80 Fuels



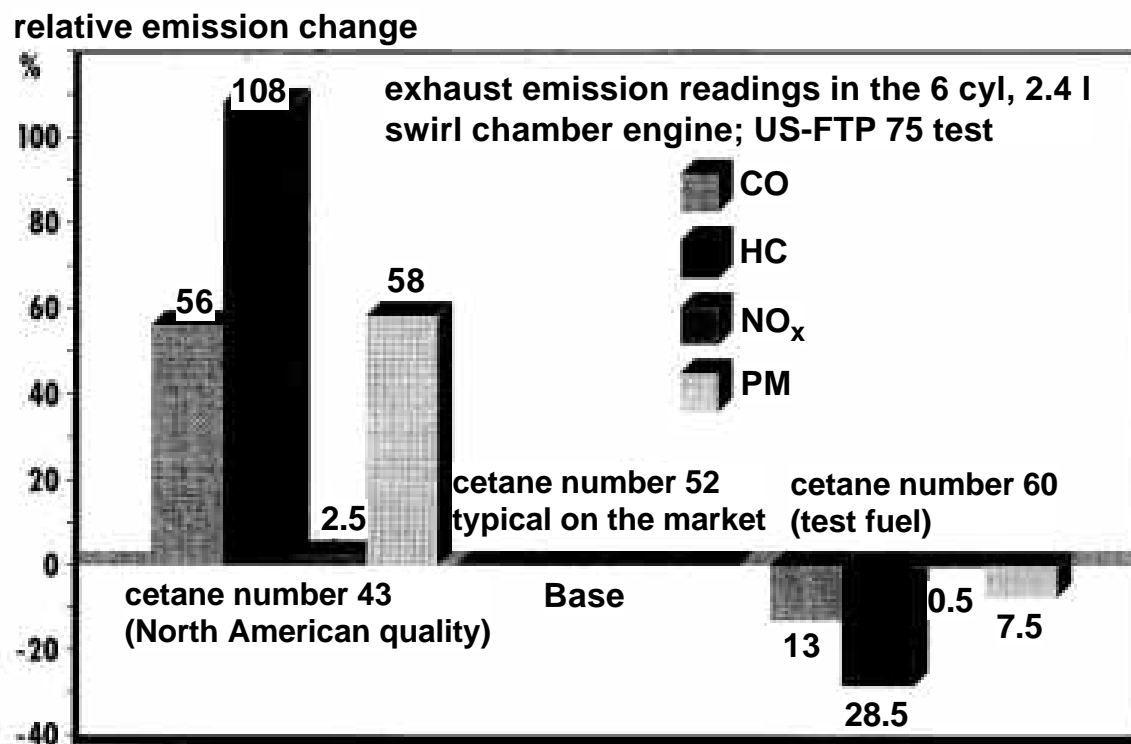
Test conditions:

2.2 l DI engine

European test cycle; MNEDC (cold test with PI)



NO_x and PM Reduction with CN 60 Fuels

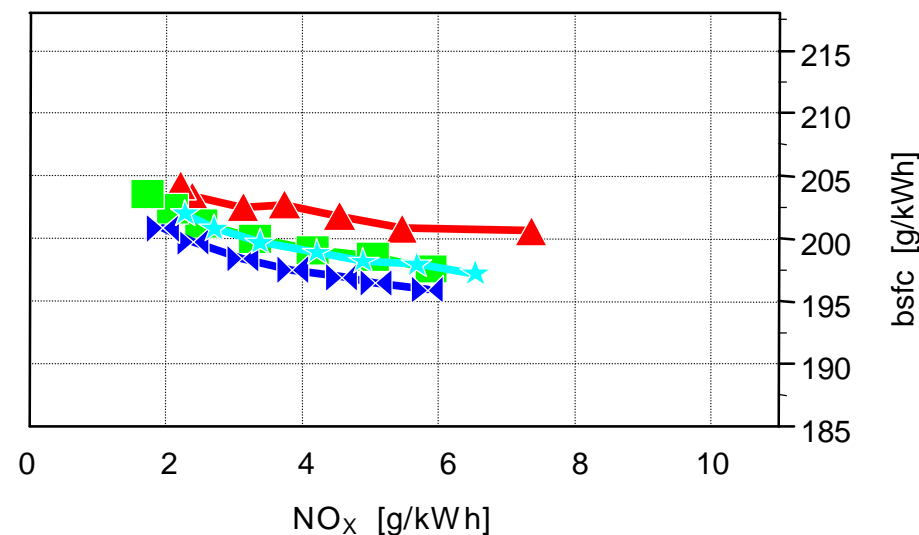
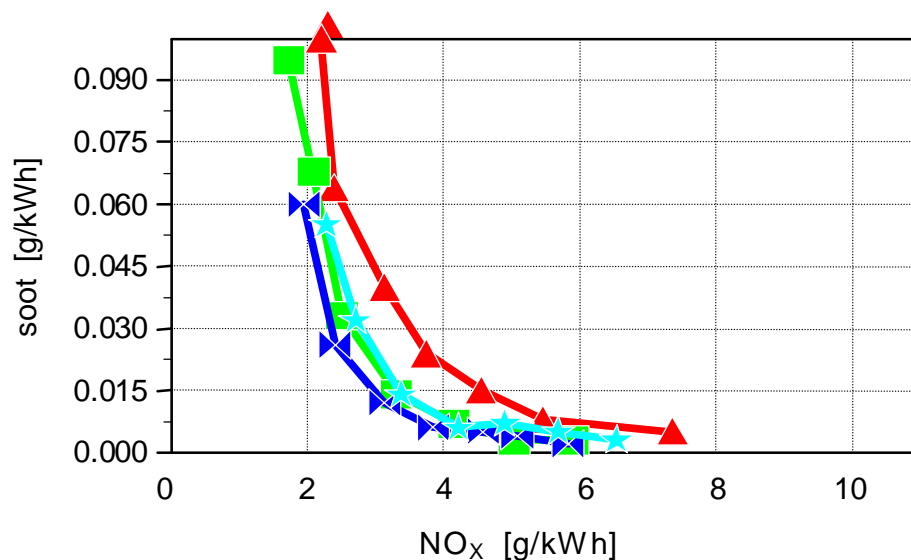


Test conditions:

6 cyl., 2.4 l, swirl chamber engine
U.S.-FTP75 test



Better Trade-offs for Soot/ NO_x and Fuel Consumption/ NO_x with CN 52 ® 59 Fuels



Test conditions:

1 Cyl. HD engine; V_d ca. 2 l, with EGR

Speed = 1710 rpm, Load = 100%

EGR rate ≤ 18 %

| | | |
|--|--------|-------------------------------|
| | fuel 1 | S = 350ppm; CN = 52; T.AH.25% |
| | fuel 4 | S = 10ppm; CN = 53; T.AH. 20% |
| | fuel 2 | S = 10ppm; CN = 55; T.AH.12% |
| | fuel 3 | S = 10ppm; CN = 59; T.AH. 7% |



Reasoning for HFRR

- HFRR is an **adequate** test method
- HFRR provides **customer satisfaction**
- **HFRR 460 µm max.** known to prevent field problems
- All high-pressure fuel-lubricated injection systems are exceedingly lubricity-sensitive
and require clean fuels (no free water and/or contamination)
- Common-rail and Rotary pumps require the same level of lubricity
- Lubricity specification in ASTM D975 needed **ASAP**
- **Spec.** should not exceed HFRR: WS1.4 £ 460 µm (ISO 12156-1)
- **Bosch and DFIE industry willing to**
 - **share and validate experience**
 - **offer more tests and**
 - **cooperate with regulators (CARB, ...)**



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